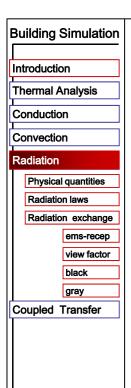


#### Lecture 3: Radiation



#### Curricula

2 x 4h Lectures

Conduction

Convection

Radiation

Coupled heat transfer

2 x 4h Tutorials and project

Model your own SmartHome

Simulate and discuss

1 x 2h Defend your project

1 x 2h Written exam

#### **Prerequisites**

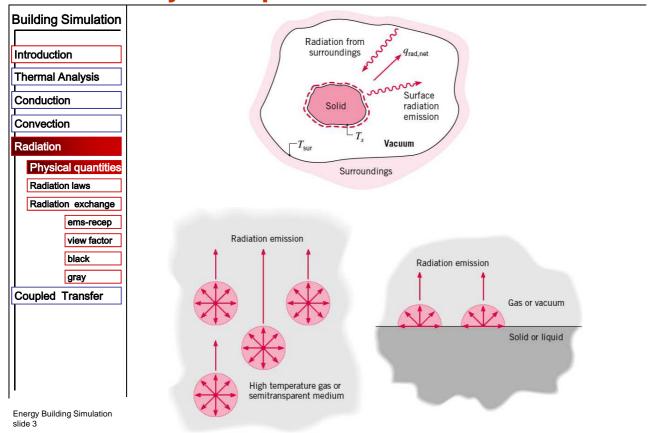
Calculus

Linear algebra

**Thermodynamics** 

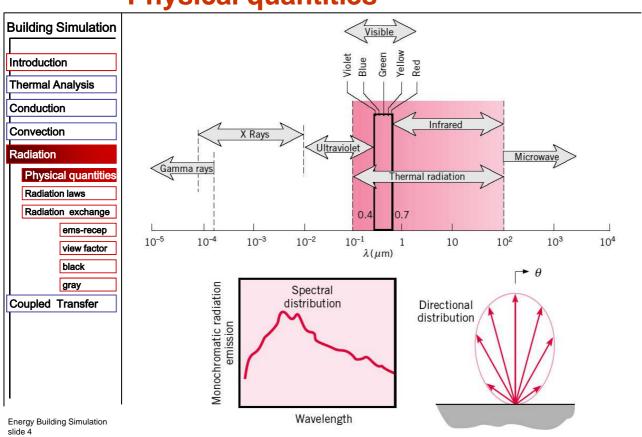
Heat and mass transfer

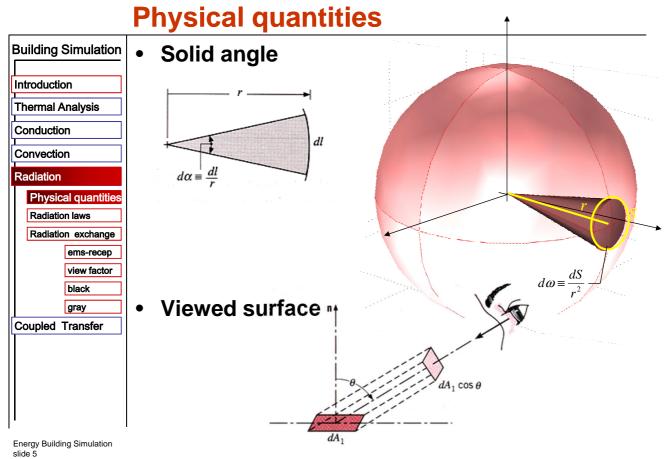
**Physical quantities** 



#### Radiation

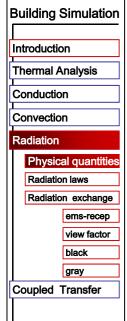
**Physical quantities** 



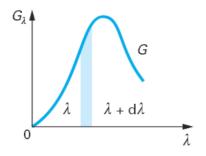


#### Radiation

# **Physical quantities**



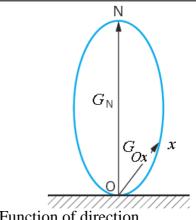
**Physical quantities** 



Function of wavelength "spectral"

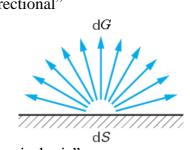
$$G_{\lambda} = \frac{dG}{d\lambda}$$

$$G = \int_0^\infty G_\lambda d\lambda$$



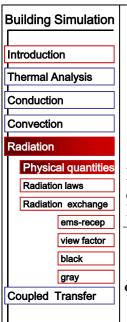
Function of direction

"directional"

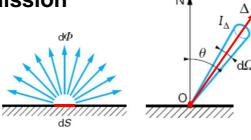


"hemispheric"

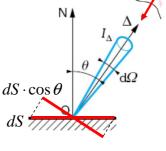
**Physical quantities** 



**Emission** 



Intensity



Luminance of the viewed surface in a direction

Flux dans tout l'espace

**Emissive** power

total

$$\Phi[W]$$
  $M = \frac{d\Phi}{dS} \left[ \frac{W}{m^2} \right]$ 

$$I_{\Delta} = \frac{d\Phi_{\Delta}}{d\Omega} \left[ \frac{\mathbf{W}}{\mathrm{sr}} \right]$$

$$I_{\Delta} = \frac{d\Phi}{dS} \left[ \frac{W}{m^2} \right] \qquad I_{\Delta} = \frac{d\Phi_{\Delta}}{d\Omega} \left[ \frac{W}{sr} \right] \qquad = \frac{d^2\Phi_{\Delta}}{d\Omega dS \cos \theta} \left[ \frac{W}{sr \cdot m^2} \right]$$

spectral

$$\Phi = \int_0^\infty \Phi_\lambda . d\lambda$$

$$M_{\lambda} = \frac{d\Phi_{\lambda}}{dS}$$

$$I_{\Delta\lambda} = \frac{d\Phi_{\lambda}}{d\Omega}$$

$$\Phi = \int_0^\infty \Phi_\lambda \cdot d\lambda \qquad M_\lambda = \frac{d\Phi_\lambda}{dS} \qquad I_{\Delta\lambda} = \frac{d\Phi_\lambda}{d\Omega} \qquad \qquad L_{\Delta\lambda} = \frac{d^2\Phi_\lambda}{d\Omega \cdot dS \cos \theta}$$

Energy Building Simulation slide 7

#### Radiation

## Physical quantities

#### **Building Simulation**

Introduction **Thermal Analysis** 

Conduction

Convection

#### Radiation

Physical quantities Radiation laws

Radiation exchange ems-recep

> view factor black

gray

Coupled Transfer

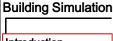
total illuminance

$$E = \frac{d\Phi}{dS} \left[ \frac{W}{m^2} \right]$$

- spectral illuminance

$$E_{\lambda} = \frac{d\Phi_{\lambda}}{dS}$$

### **Physical quantities**



Introduction Thermal Analysis

Conduction Convection

#### Radiation

Physical quantitie Radiation laws Radiation exchange

ems-recep view factor

Coupled Transfer

gray

#### • Relation emissive power - illuminance

Heat Flux directional  $d^2\Phi_{Ox} = L_{Ox} \cdot dS \cos\theta \cdot d\Omega$ 

Hemispheric flux

$$d\Phi = L \cdot dS \int_{S} \cos \theta \cdot d\Omega$$

$$= L \cdot dS \int_C ds'$$
$$= L \cdot dS \cdot \pi$$

$$\frac{d\Phi}{dS} = L \cdot \pi$$

Relation emissive power - illuminance

$$M = \pi L$$

Energy Building Simulation slide 9

#### Radiation

### **Physical quantities**

#### **Building Simulation**

Introduction **Thermal Analysis** 

Conduction Convection

#### Radiation

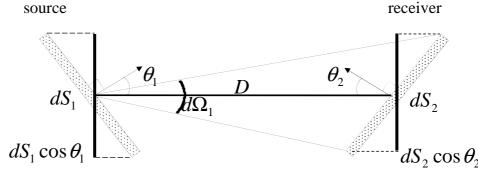
Physical quantitie Radiation laws Radiation exchange

> ems-recep view factor black

gray

Coupled Transfer

#### Relation between physical quantities: de Bouguer



$$d^{2}\Phi_{12} = L_{1} \cdot dS_{1} \cos \theta_{1} \cdot d\Omega_{1} \qquad d\Omega_{1} = \frac{dS_{2} \cos \theta_{2}}{D^{2}}$$

$$d\Omega_1 = \frac{dS_2 \cos \theta_2}{D^2}$$

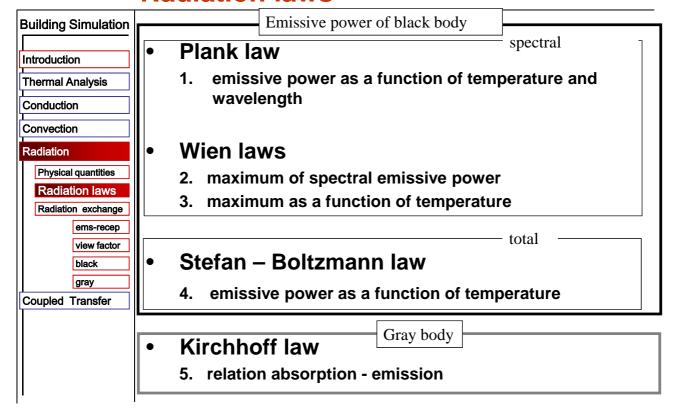
 $ds = d\Omega \cdot R^2$ 

 $= d\Omega$ 

 $ds' = ds \cdot \cos \theta$ 

$$dE = \frac{d^2 \Phi_{12}}{dS_2} = L_1 \frac{\cos \theta_1 \cdot \cos \theta_2}{D^2} dS_1$$

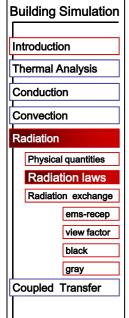
#### **Radiation laws**



Radiation

**Energy Building Simulation** 

#### **Radiation laws**



Black body:

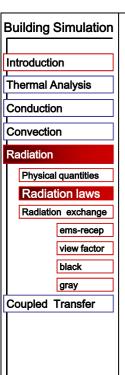
- absorbs all incident radiation (regardless of wavelength or direction)
- emits the maximum energy for a given temperature and wavelength
- is a diffuse emitter (but in function of temperature and wavelength) according to Lambert law

$$L^0 = rac{M^0}{\pi}$$
  $L^0_\lambda = rac{M^0_\lambda}{\pi}$ 

Black body is:

- reference for real bodies: "yardstick" for radiation
- perfect "source" and "absorber"
- characterized by hemispheric physical quantities (noted °)

#### **Radiation laws**



Plank law

• Plank law
$$M^{\circ}_{\lambda,T} = \frac{C_1 \lambda^{-5}}{\frac{C_2}{\lambda T} - 1}$$

$$C_1 = 2\pi hc^2 = 3.74 \cdot 10^{-16} [\text{W} \cdot \text{m}^2]$$

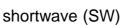
$$C_2 = hc/k = 1.4 \cdot 10^{-2} [\text{m} \cdot \text{K}]$$

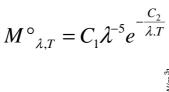
$$c \text{ speed of light}$$

$$h \text{ Plank constant}$$

$$C_1 = 2\pi hc^2 = 3,74 \cdot 10^{-16} [\text{W} \cdot \text{m}^2]$$
  
 $C_2 = hc/k = 1,4 \cdot 10^{-2} [\text{m} \cdot \text{K}]$ 

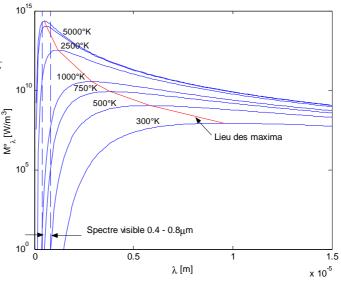
- h Plank constant
- Boltzmann constant





longwave (LW)

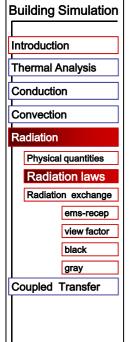
$$M^{\circ}_{\lambda,T} = \frac{C_1 T}{C_2 \lambda^4}$$



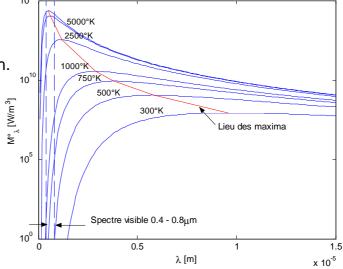
Energy Building Simulation slide 13

#### Radiation

#### **Radiation laws**

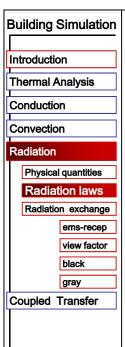


- Plank law
- 1. Emissive power varies with wavelength.
- 2. Emissive power increases with the temperature of the source for every wavelength.
- 3. Visible spectrum contains a large majority of solar radiation.



4. Effective emission band depends on source temperature: higher the temperature, higher the frequency radiation (Wien laws, separation SW / LW).

#### **Radiation laws**



Wien laws

 $\lambda$  for maximum spectral power emission

$$\frac{dM_{\lambda T}^{0}}{d\lambda} = 0 \Rightarrow \lambda_{M}T = 2.897 \cdot 10^{-3} [\text{m} \cdot \text{K}]$$

2. maximum spectral power emission

$$M_{\lambda_u T}^0 = B \cdot T^5$$
;  $B = 1.286 \cdot 10^5 [\text{W} \cdot \text{m}^{-3} \cdot \text{K}^{-5}]$ 

Stefan - Boltzmann law

$$M^{0} = \int_{0}^{\infty} M_{\lambda T}^{0} d\lambda = \sigma_{0} T^{4}$$

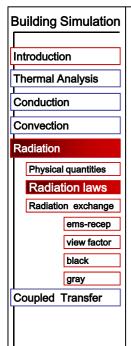
$$M^{0} = 5.68 \left(\frac{T}{100}\right)^{4}$$

$$\sigma_0 = 5.68 \cdot 10^{-8} \text{ W m}^{-2} \text{ K}^{-4}$$

Energy Building Simulation slide 15

#### Radiation

#### **Radiation laws**



Effective emission band

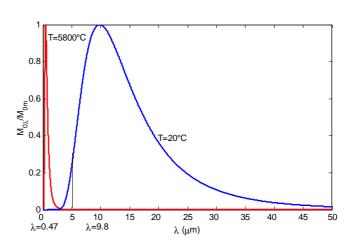
Fraction of total emission in a band  $F_{\lambda_1-\lambda_2} = \frac{\int_{\lambda_1}^{\lambda_2} M_{\lambda T}^0 d\lambda}{M^0}$ 

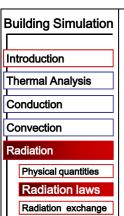
**Effective spectral band** 

 $F_{0.5\lambda-5\lambda} = 0.956$ 

solar emission: 50% visible, 40% IR, 8% UV<sup>\*</sup>

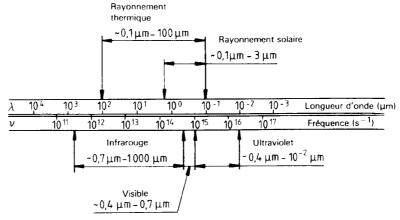
SW and LW radiation





ems-recep
view factor
black
gray

Coupled Transfer



Emissive power of black body

Temperature		Emissive power	Max. wavelength	Spectral band
Absolue, T	Celsius, $\theta$	$M^{0}$	$\lambda_{\scriptscriptstyle M}$	$0.5\lambda_{\scriptscriptstyle M} - 5\lambda_{\scriptscriptstyle M}$
[K]	[°C]	$[W/cm^2]$	[ <i>μ</i> m]	$[\mu \mathrm{m}]$
300	27	0.05	9.6	4.8 - 41
500	227	0.36	5.7	3.0 - 25
750	477	1.80	3.8	2.0 - 16
1000	727	5.70	2.9	1.5 - 12
1200	927	11.82	2.4	1.2 - 11
1500	1227	28.90	1.9	1.0 - 8
2000	1727	91.00	1.4	0.7 - 6
3000	2727	462.00	0.96	0.5 - 4
5790	5517	6383.6	0.50	0.25 - 2.5

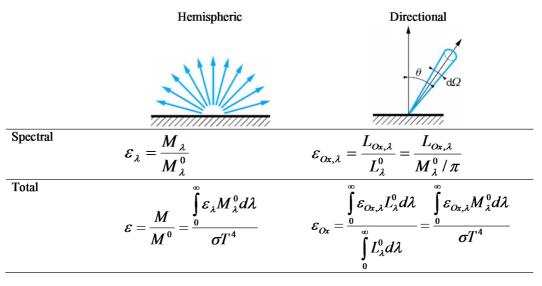
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#### Radiation

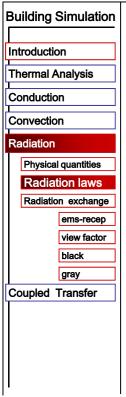
#### **Radiation laws**

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gray
Coupled Transfer

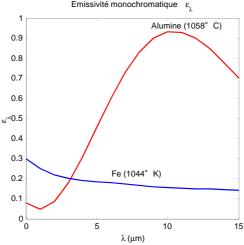
Emission from real surfaces: comparison with a black body for the same temperature and wavelength = emissivity

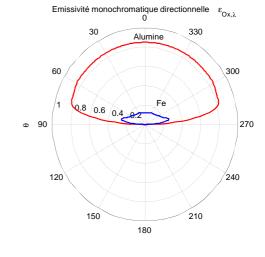


#### **Radiation laws**



#### Emissivity





spectral, hemispherical

$$\varepsilon_{\lambda} = \frac{M_{\lambda}}{M_{\lambda}^{0}}$$

spectral, directional

$$\varepsilon_{Ox,\lambda} = \frac{L_{Ox,\lambda}}{L_{\lambda}^{0}} = \frac{L_{Ox,\lambda}}{M_{\lambda}^{0}/\pi}$$

Energy Building Simulation slide 19

#### Radiation

### **Radiation laws**

**Building Simulation** 

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Coupled Transfer

Emissivity: particular cases

Gray body

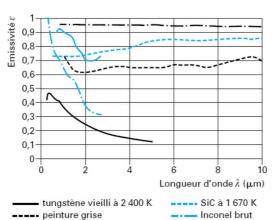
peinture noire crown

- Diffuse (isotropic) emission
- Gray and diffuse

 $\mathcal{E}_{Ox,\lambda} \to \mathcal{E}_{Ox}$ ;  $\mathcal{E}_{\lambda} \to \mathcal{E}$ 

$$\mathcal{E}_{Ox,\lambda} \to \mathcal{E}_{\lambda}; \ \mathcal{E}_{Ox} \to \mathcal{E}$$

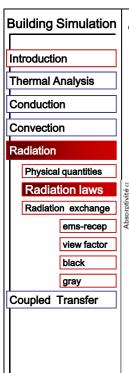
$$\varepsilon_{Ox,\lambda} \to \varepsilon$$



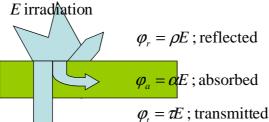
Inconel oxydé

Example of emissivity

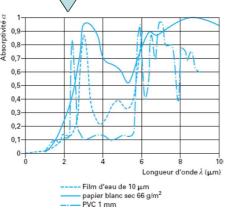
#### **Radiation laws**

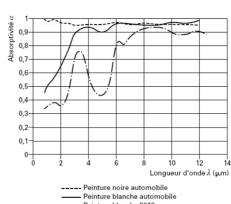


**Radiation reception** 



 $\alpha + \tau + \rho = 1$ 





— Peinture blanche automobile — Peinture blanche 9010

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#### Radiation

#### **Radiation laws**

**Building Simulation** 

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> view factor black

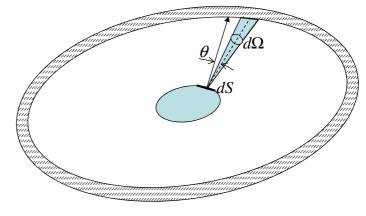
> > gray

ems-recep

Coupled Transfer

Kirchhoff law: relation emissivity and absorptivity

$$\varepsilon_{Ox,\lambda} = \alpha_{Ox,\lambda}$$



emission:

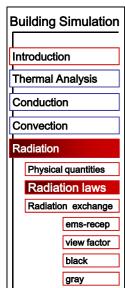
$$\left(d^2\Phi_{Ox,\lambda}\right)_e = \varepsilon_{Ox,\lambda} L_{\lambda}^0 dS \cos\theta d\Omega$$

absorption:

$$\left(d^2\Phi_{Ox,\lambda}\right)_a = \alpha_{Ox,\lambda} L_{\lambda}^0 dS \cos\theta d\Omega$$

**Energy Building Simulation** 

#### **Radiation laws**



Kirchhoff law

 $^{-}$   $\varepsilon_{\scriptscriptstyle Ox,\lambda}$  =  $\alpha_{\scriptscriptstyle Ox,\lambda}$ 

diffuse emission

 $\varepsilon_{\lambda} = \alpha_{\lambda}$ 

in general

 $\varepsilon \neq \alpha$ 

 $\varepsilon(T) = \frac{M(T)}{M^{0}(T)} = \frac{\int_{0}^{\infty} \varepsilon_{\lambda} M_{\lambda}^{0}(T) d\lambda}{\int_{0}^{\infty} M_{\lambda}^{0}(T) d\lambda} = \frac{\int_{0}^{\infty} \varepsilon_{\lambda} M_{\lambda}^{0}(T) d\lambda}{\sigma T^{4}}$  (own temp.)

 $\alpha = \frac{\varphi_a}{E} = \frac{\int_0^\infty \alpha_\lambda E_\lambda d\lambda}{\int_0^\infty E_\lambda d\lambda}$  (depends on received radiation)

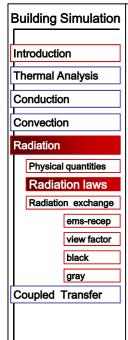
- exceptions
  - gray bodies  $\varepsilon = \alpha$
  - black body  $\varepsilon = \alpha = 1$

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Coupled Transfer

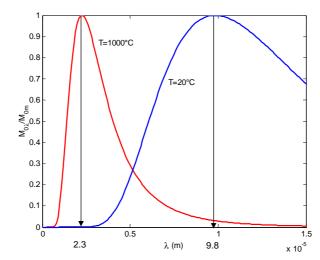
#### Radiation

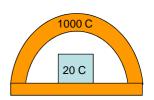
#### **Radiation laws**



• Practical consequences: radiative heating

Radiation in different spectral bands  $F_{0.5\lambda-5\lambda} = 0.956$ 





#### **Radiation laws**

#### **Building Simulation**

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gray

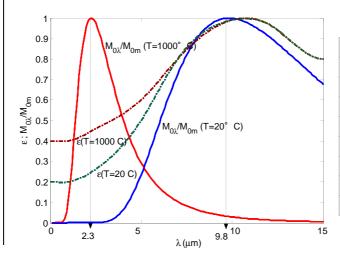
Coupled Transfer

Practical consequences: radiative heating

for every surface,  $\varepsilon$  corresponding to a spectral band

$$\varepsilon_{20^{\circ}C} = \frac{1}{50 - 5} \int_{5}^{50} \varepsilon_{\lambda} \cdot d\lambda = 0.9$$

$$\varepsilon_{20^{\circ}C} = \frac{1}{50 - 5} \int_{5}^{50} \varepsilon_{\lambda} \cdot d\lambda = 0.9$$
  $\varepsilon_{1000^{\circ}C} = \frac{1}{5 - 1.25} \int_{1.25}^{5} \varepsilon_{\lambda} \cdot d\lambda = 0.43$ 



	5790 <i>K</i>	300 <i>K</i>	
Matériau	a	ε	
Carton goudronné noir	0,82	0,91	
Brique rouge	0,75	0,93	
Blanc de zinc	0,22	0,92	
Neige propre	0,200,35	0,95	
Chrome poli	0,40	0,07	
Or poli	0,29	0,026	
cuivre poli	0,18	0,03	
cuivre, oxydé	0,70	0,45	

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#### Radiation

#### **Radiation laws**

#### **Building Simulation**

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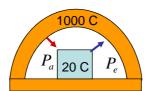
Radiation exchange ems-recep

> view factor black

gray

Coupled Transfer

Practical consequences: radiative heating



$$P_a = \varepsilon_{1000^{\circ}C} M^0 = 0.43 \sigma (1273)^4 = 6.7 \cdot 10^4 \text{ W/m}^2$$

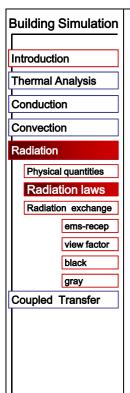
$$P_e = \varepsilon_{20^{\circ}C} M^0 = 0.9 \sigma (293)^4 = 3.76 \cdot 10^2 \text{ W/m}^2$$

 $P_a > P_e \implies \text{body is heating}$ 

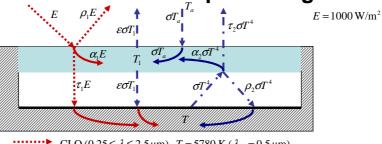
**Temperature** 

- source of radiation
- own

#### **Radiation laws**



Practical consequences: greenhouse effect



CLO  $(0.25 \le \lambda \le 2.5 \,\mu\text{m})$ ,  $T = 5780 \,\text{K} \,(\lambda_M = 0.5 \,\mu\text{m})$ 

**T** GLO(3 ≤  $\lambda$  ≤ 30 µm),T = 450 K ( $\lambda_M$  = 6.1 µm)

GLO  $(5 \le \lambda \le 50 \,\mu\text{m}), T = 300 \,\text{K} (\lambda_M = 9.8 \,\mu\text{m})$ 

Bilan sur la vitre

$$2\varepsilon \cdot \sigma T_1^4 = \alpha_1 E + \alpha_2 \sigma T^4 + \alpha_3 \sigma T_a^4$$

Rayonnement absorbé:

 $\alpha_{1}E$ -solaire

 $\alpha_2 \sigma T^4$  - émis par la surface noire

 $\alpha_3 \sigma T_a^4$  -émis par l'environnement

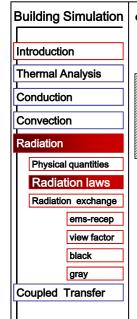
Rayonnement émis:

 $2\varepsilon \cdot \sigma T_1^4$  - les deux surfaces de la vitre

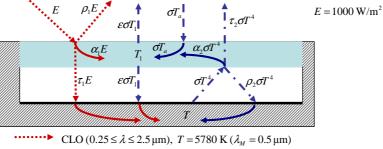
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#### Radiation

#### **Radiation laws**



Practical consequences: greenhouse effect



• GLO (3  $\leq \lambda \leq$  30 µm),  $T = 450 \text{ K} (\lambda_M = 6.1 \text{ µm})$ 

→ GLO (5 ≤  $\lambda$  ≤ 50 µm), T = 300 K ( $\lambda_M$  = 9.8 µm)

Balance on black absorber

Absorbed radiation:

 $\tau_{1}E$ transmitted solar radiation through the glass

emmitted by the glass

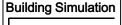
 $\rho_2 \sigma T^4$  reflected by the glass

 $\sigma T^4 = \tau_1 E + \varepsilon \sigma T_1^4 + \rho_2 \sigma T^4$ 

Emmitted radiation:

 $\sigma T^4$  - towards the glass

#### **Radiation laws**



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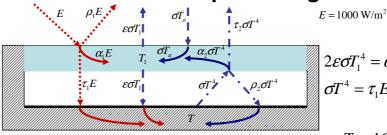
Radiation exchange ems-recep

gray

view factor

Coupled Transfer

Practical consequences: greenhouse effect



 $2\varepsilon\sigma T_1^4 = \alpha_1 E + \alpha_2 \sigma T^4 + \alpha_3 \sigma T_a^4$ 

 $\sigma T^4 = \tau_1 E + \rho_2 \sigma T^4 + \varepsilon \sigma T_1^4$ 

 $\Rightarrow T = 460 \text{ K} = 187 \,^{\circ}\text{C}$ 

CLO  $(0.25 \le \lambda \le 2.5 \,\mu\text{m})$ ,  $T = 5780 \,\text{K} (\lambda_M = 0.5 \,\mu\text{m})$ GLO  $(3 \le \lambda \le 30 \,\mu\text{m})$ ,  $T = 450 \,\text{K} (\lambda_M = 6.1 \,\mu\text{m})$ 

- → GLO (5 ≤  $\lambda$  ≤ 50 μm), T = 300 K ( $\lambda_M$  = 9.8 μm)

Radiative properties of glass

	Spectral band	Temperature	α	ρ	τ
1	$0.25 \le \lambda \le 2.5 \mu\text{m}$	5780 K ( $\lambda = 0.5 \mu\text{m}$ )	0	0.05	0.95
2	$3 \le \lambda \le 30 \mu m$	450 K ( $\lambda = 6.1 \mu m$ )	0.65	0.30	0.05
3	$5 \le \lambda \le 50 \mu\text{m}$	$300 \text{ K} (\lambda = 9.8 \mu\text{m})$	1.00	0	0

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#### Radiation

#### **Radiation laws**

#### **Building Simulation**

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Radiation exchange

ems-recep

black

gray

Coupled Transfer

#### Radiation laws

#### • Practical consequences: greenhouse effect

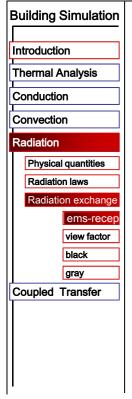
Propriétés radiatives du verre ρ Température de Bande  $\alpha$  $\tau$ spectrale rayonnement 1  $0.25 \le \lambda \le 2.5 \,\mu\text{m}$ 0.95 5780 K ( $\lambda = 0.5 \,\mu m$ ) 0 0.05 2  $5 \le \lambda \le 50 \ \mu m$  $300 \text{ K} (\lambda = 9.8 \, \mu\text{m})$ 1.00 0 0



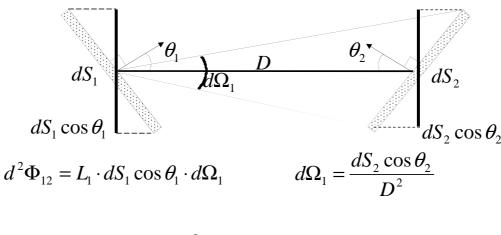




### Radiation exchange: emission-reception



• Bouguer relation

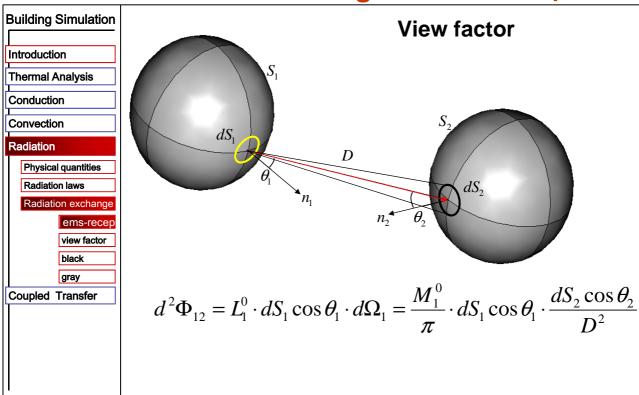


$$d^2\Phi_{12} = \frac{M_1^0}{\pi} \cdot dS_1 \cos \theta_1 \cdot \frac{dS_2 \cos \theta_2}{D^2}$$

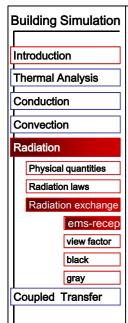
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#### Radiation

#### Radiation exchange: emission-reception



#### Radiation exchange: emission-reception



$$\Phi_{12} = M_1^0 \int_{S_1} \int_{S_2} \frac{dS_1 \cos \theta_1 \cdot dS_2 \cos \theta_2}{\pi \cdot D^2}$$

View factor

$$F_{12} = \frac{\Phi_{12}}{\Phi_{1}} = \frac{\Phi_{12}}{M_{1}^{0}S_{1}} = \frac{1}{\pi S_{1}} \int_{S_{1}S_{2}} \frac{dS_{1}\cos\theta_{1} \cdot dS_{2}\cos\theta_{2}}{D^{2}}$$

$$F_{21} = \frac{\Phi_{21}}{\Phi_2} = \frac{1}{\pi S_2} \int_{S_1 S_2} \frac{dS_1 \cos \theta_1 \cdot dS_2 \cos \theta_2}{D^2}$$

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#### Radiation

### Radiation exchange: view factors

#### **Building Simulation**

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view fac

gray

Coupled Transfer

- Relations between view factors
  - reciprocity
  - complementarity(closed enclosure)

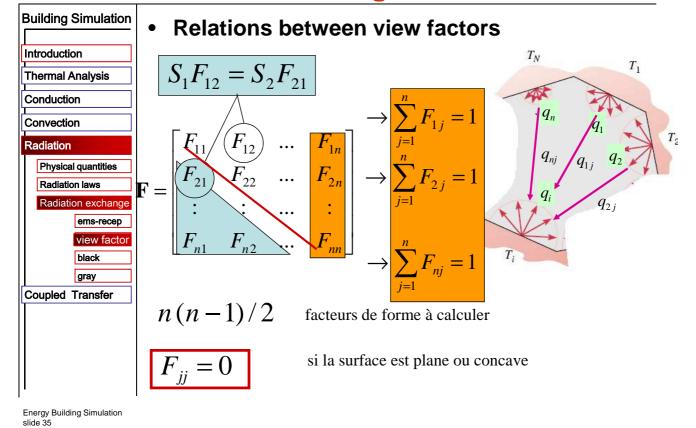
$$S_1 F_{12} = S_2 F_{21}$$

$$\sum_{i=1}^{n} F_{ij} = 1$$

$$\Phi_i = \sum_{j=1}^n \Phi_{ij}$$

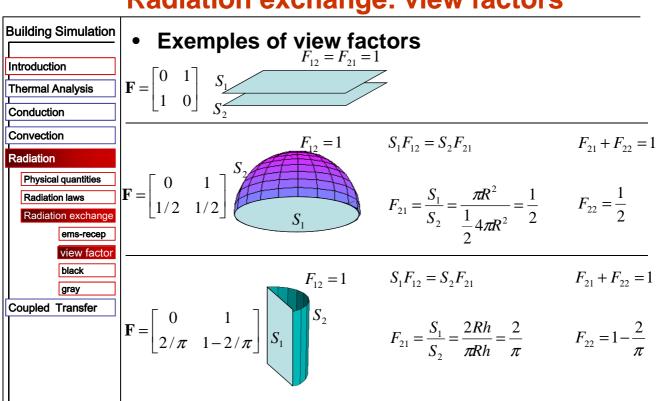
$$\Phi_i = \sum_{i=1}^n F_{ij} \Phi_i$$

#### Radiation exchange: view factors



#### Radiation

# Radiation exchange: view factors



### Radiation exchange: view factors



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view factor

Coupled Transfer

#### Two small surfaces

$$d^2\Phi_{12} = L_1^0 \cdot dS_1 \cos \theta_1 \cdot d\Omega_1 = \frac{M_1^0}{\pi} \cdot dS_1 \cos \theta_1 \cdot \frac{dS_2 \cos \theta_2}{D^2}$$

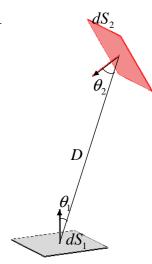
$$\Phi_{12} = M_1^0 \frac{dS_1 \cos \theta_1 \cdot dS_2 \cos \theta_2}{\pi D^2}$$
 $\Phi_1 = M_1^0 \cdot dS_1$ 

$$\Phi_1 = M_1^0 \cdot dS_1$$

$$F_{12} \equiv \frac{\Phi_{12}}{\Phi_{1}} = \frac{\Phi_{12}}{M_{1}^{0} \cdot dS_{1}} = \frac{\cos \theta_{1} \cdot \cos \theta_{2}}{\pi D^{2}} dS_{2}$$

$$F_{21} = \frac{\cos \theta_1 \cdot \cos \theta_2}{\pi D^2} dS_1$$

$$\mathbf{F} = \begin{bmatrix} 0 & \frac{\cos\theta_1 \cdot \cos\theta_2}{\pi D^2} dS_2 \\ \frac{\cos\theta_1 \cdot \cos\theta_2}{\pi D^2} dS_1 & 0 \end{bmatrix}$$



**Energy Building Simulation** 

#### Radiation

### Radiation exchange: black surfaces

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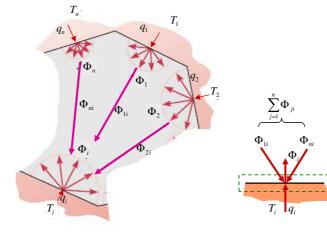
Physical quantities Radiation laws

Radiation exchange

ems-recep view factor

gray Coupled Transfer

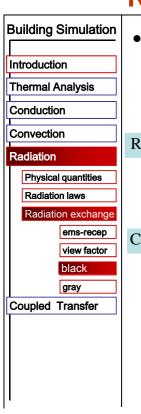
- Net radiative heat exchange
  - exchange in a closed black enclosure



$$q_{i} \equiv \Phi_{i,net} = \Phi_{i} - \sum_{j=1}^{n} \Phi_{ji} = \Phi_{i} - \sum_{j=1}^{n} F_{ji} \Phi_{j} = S_{i} M_{i}^{0} - \sum_{j=1}^{n} S_{j} F_{ji} M_{j}^{0}$$
net sent reçeived

Black surfaces

#### Radiation exchange: black surfaces



Net heat

- black, closed enclosure

$$S_{i}F_{ij} = S_{j}F_{ji} \Rightarrow \Phi_{i,net} = S_{i}M_{i}^{0} - \sum_{j=1}^{n} S_{j}F_{ji}M_{j}^{0} = S_{i}M_{i}^{0} - S_{i}\sum_{j=1}^{n} F_{ij}M_{j}^{0}$$
Réciprocité met émis reçu

$$\sum_{j=1}^{n} F_{ij} = 1 \qquad \Rightarrow \Phi_{i,net} = S_i M_i^0 - S_i \sum_{j=1}^{n} F_{ij} M_j^0 = S_i M_i^0 \sum_{j=1}^{n} F_{ij} - S_i \sum_{j=1}^{n} F_{ij} M_j^0$$

$$\Phi_{i,net} = \sum_{i=1}^{n} S_i F_{ij} (M_i^0 - M_j^0)$$

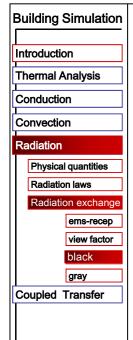
flux échangé entre les surfaces noires  $S_i \rightarrow S_j$ 

$$\Phi_{i,net} = \sum_{j=1}^{n} S_{i} F_{ij} (M_{i}^{0} - M_{j}^{0}) = \sum_{j=1}^{n} (S_{i} F_{ij} M_{i}^{0} - S_{j} F_{ji} M_{j}^{0}) = \sum_{j=1}^{n} \Phi_{ij,net}$$
émis reçu net

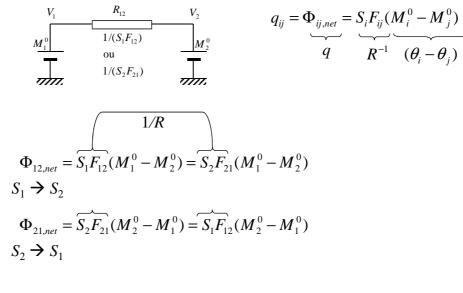
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#### Radiation

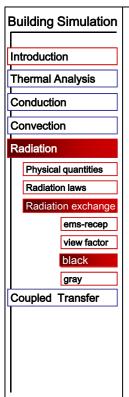
### Radiation exchange: black surfaces



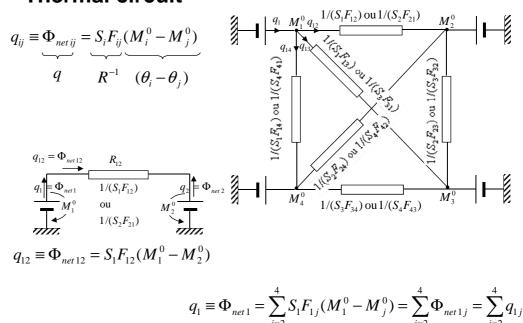
Thermal circuit



## Radiation exchange: black surfaces



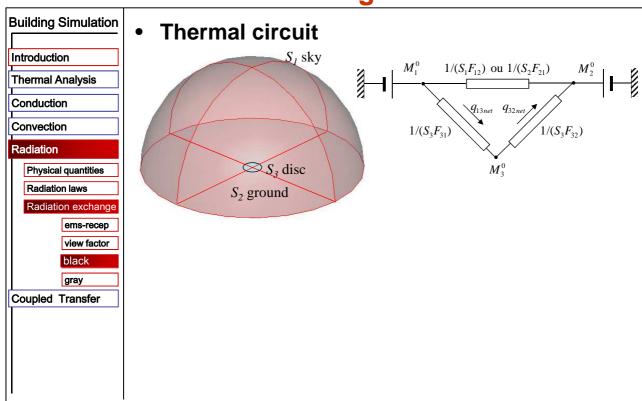
#### Thermal circuit



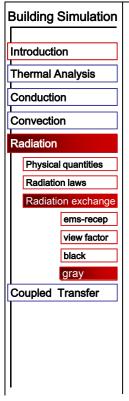
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#### Radiation

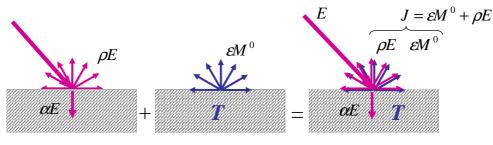
### Radiation exchange: black surfaces



### Radiation exchange: gray surfaces



#### Radiosité



$$J = \varepsilon M^{0} + \rho E$$
Opaque surface  $\tau = 0$ 

$$\Rightarrow \rho = 1 - \alpha = 1 - \varepsilon$$

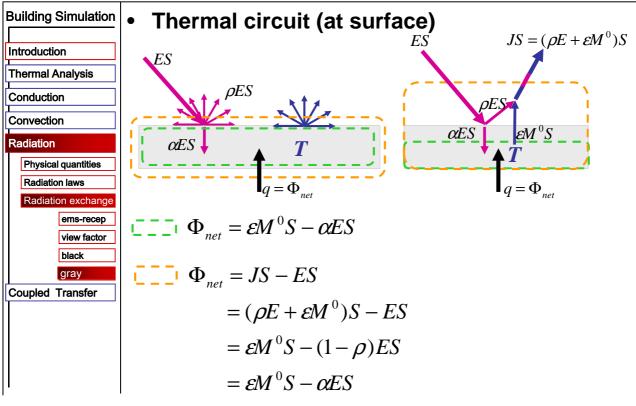
$$\Rightarrow J = \varepsilon M^{0} + (1 - \varepsilon)E$$
equal if the same wavelength!

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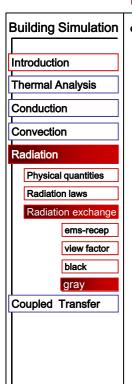
or gray body

#### Radiation

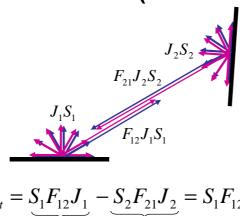
#### Radiation exchange: gray surfaces



### Radiation exchange: gray surfaces



• Thermal circuit (between surfaces)

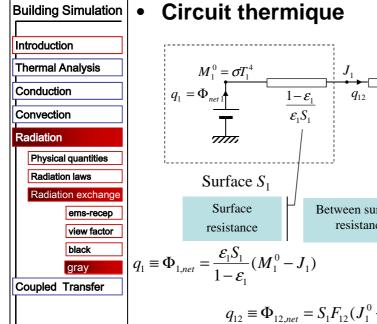


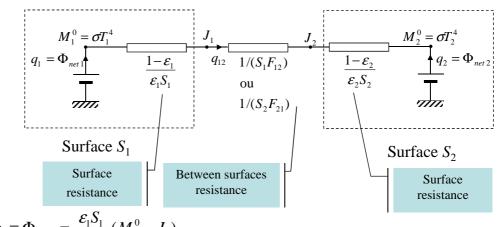
$$\underbrace{\Phi_{12,net}}_{\text{net}} = \underbrace{S_1 F_{12} J_1}_{\text{émis}} - \underbrace{S_2 F_{21} J_2}_{\text{reçu}} = S_1 F_{12} (J_1 - J_2) = S_2 F_{21} (J_1 - J_2)$$

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#### Radiation

### Radiation exchange: gray surfaces





$$\begin{aligned} q_{12} &\equiv \Phi_{12,net} = S_1 F_{12} (J_1^0 - J_2^0) = S_2 F_{21} (J_1^0 - J_2^0) \\ q_2 &\equiv \Phi_{2,net} = \frac{\varepsilon_2 S_2}{1 - \varepsilon_2} (M_2^0 - J_2) \end{aligned}$$

## Radiation exchange: gray surfaces

