

Exercise EHL

This exercise studies the lubrication of a circular contact (ball bearing in gear box) under the following conditions ::

Lubricant : $\alpha = 2 \cdot 10^{-8} \text{ Pa}^{-1}$, $\eta_0 = 2 \cdot 10^{-1} \text{ Pa.s}$,

Material : $E' = 2 \cdot 10^{11} \text{ Pa}$,

Geometry : $R = 0.01 \text{ m}$,

Speed : $u_1 + u_2 = 1.0 \text{ m/s}$

Load : $w = 20 \text{ N}$

We start our study by a Hertzian (dry contact) analysis

1) Compute the maximum pressure and contact radius p_h and a .

$p_h = 0.73 \text{ GPa}$, $a = 1.14 \cdot 10^{-4} \text{ m}$

2) Compute the operating parameters W , U et G .

$W = 1 \cdot 10^{-6}$, $U = 1 \cdot 10^{-10}$, $G = 4000$

3) compute the Moes parameters, indicate the lubrication regime and read the film thickness from the graph, convert the value to microns

$M = 31.6$, $L = 12.6$: piezo-viscous-elastic regime, $H_c = 4.5$, $h_c = 0.45 \text{ micrometer}$

In the next part the circular contact is approximated by a line contact ($a=b$).

4) choose w_1 such that the line contact and circular contact operate under the same (piezoviscous) lubrication conditions.

$a=b=0.114 \text{ mm}$, p_h identique! $p_h = 0.73 \text{ GPa}$, $w_1 = 1.3 \cdot 10^5 \text{ N/m}$

5) what is the total load on the line contact assuming its width to be $2a$.

$W = 29.6 \text{ N}$, not much different from the 20 N of the circular contact!